

# Recap of Thermodynamics Ideas discussed week 2

Open vs Closed thermodynamic system

mass flow

no mass flow

Modes of Heat transfer

$$\text{Conduction} \quad \text{Heat [J]} \quad \text{cross sectional area } [m^2]$$

rate of heat flow  $\left[ \frac{dQ}{dt} \right] = -k_p A \frac{dT}{dx}$  [temperature] thermal gradient

$\uparrow$   $\frac{dT}{dx}$  position through material

thermal conductivity  $\left[ \frac{W}{mK} \right]$  engineering tool by x

is a good resource as is CRC handbook of Physics and Chemistry

Resistance to heat flow

$$R = \frac{L}{k_p} \quad \begin{matrix} \leftarrow \text{thickness of material} \\ \leftarrow \text{thermal conductivity} \end{matrix}$$

$$\left[ \frac{m^2 K}{W} \right]$$

$$\left[ \frac{ft^2 F}{BTU/hr} \right]$$

- BTU ≡ heat to warm 1 lb of H<sub>2</sub>O 1°F

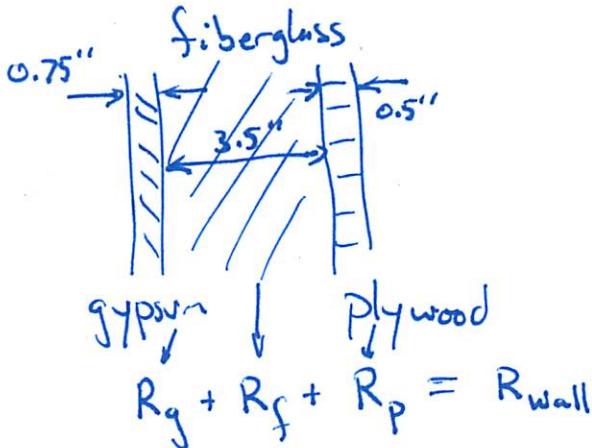
larger R ⇒ better insulator  
either thicker insulation  
or lower thermal conduction

↑  
implicit units for R or const. materials in US.

# Efficient Conversion of $k_T$ from SI $\rightarrow$ Imperial

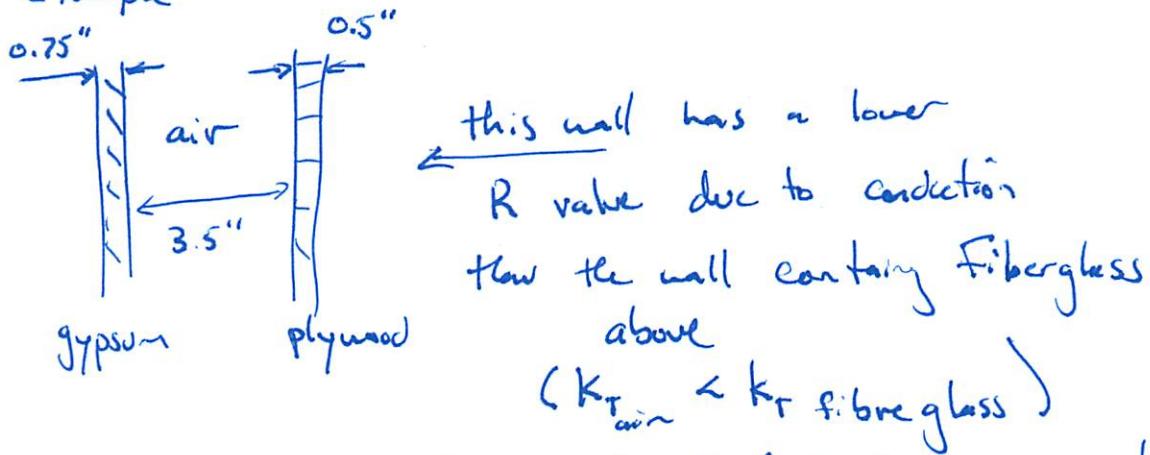
$$\left| \frac{W}{mK} \right| \times \frac{1 \text{ BTU}}{1055 \text{ Ws}} \times \frac{3600 \text{ s}}{\text{hr}} \times \frac{5^\circ\text{C}}{9^\circ\text{F}} \times \frac{1 \text{ m}}{3.28 \text{ ft}} = 0.5778 \frac{\text{BTU}/\text{hr}}{\text{ft}^\circ\text{F}}$$

Note that R-values of an assembly add



Note that the value of  $R$  can be misleading if other modes of heat transfer are relevant

Example



But rate of Heat Flow is greater w/ air

due to convection

Convection

$$\frac{dQ}{dt} = h A \frac{\text{area of surface}}{\text{temp of surface}} (T_s - T_{\infty})$$

air  
temp far from surface

Convection Coeff.